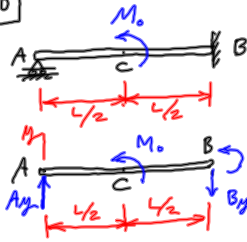


9.50



Unknowns:  $A_y$ ,  $C_1$ , &  $C_2$

- B.C. ①  $y(0) = 0$  : .....  
 ②  $y(L) = 0$  : .....  
 ③  $y'(L) = 0$  : .....

$\vec{A} = ?$   $y_c = ?$

$q = A_y \langle x \rangle^1 - M_0 \langle x - \frac{L}{2} \rangle^{-2}$

$V = A_y \langle x \rangle^0 - M_0 \langle x - \frac{L}{2} \rangle^{-1}$

$EI y'' = M = A_y \langle x \rangle^1 - M_0 \langle x - \frac{L}{2} \rangle^0$

$EI y' = \frac{A_y}{2} \langle x \rangle^2 - M_0 \langle x - \frac{L}{2} \rangle^1 + C_1$

$EI y = \frac{A_y}{6} \langle x \rangle^3 - \frac{M_0}{2} \langle x - \frac{L}{2} \rangle^2 + C_1 x + C_2$

$C_2 = 0, C_1 = 0, A_y = 0$

$\vec{A} = \vec{A}_y = 0 \uparrow$   $y_c = y(\frac{L}{2}) = 0$

Method using moment-area theorems

1st moment-area theorem:

$\theta_{D/C} = A_{C/D}$

$\theta_{D/C} = \theta_D - \theta_C$

Note that the point D of the beam must be on the right side of the beam, and  $A_{C/D}$  is the algebraic value of the elastic weight diagram between points C & D.

2nd moment-area theorem:

$t_{D/C} = (M_D)_{C/D}$

Note that  $t_{D/C}$  is the tangential deviation of point D with respect to the tangent drawn at point C.

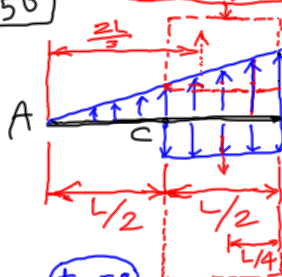
$t_{D/C}$  = vertical displacement from point D of the deflected beam to the tangent drawn at point C.

The sign of  $(M_D)_{C/D}$  must be consistent with the direction of the moment of  $\vec{F}_{D/C}$  with respect to C

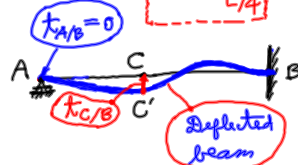
Elastic weight =  $\frac{M}{EI}$

9.50

Elastic weight between C & B



$\frac{M}{EI}$  diagram drawn by parts



Beam deflected by  $\vec{M}_0$  at C

$t_{A/B} = (M_A)_{A/B} = 0$

$\frac{2}{3} \cdot \frac{L}{2} \cdot \frac{3L}{8} \cdot \frac{A_y L}{EI} - \frac{3L}{4} \cdot \frac{M_0}{EI} \cdot \frac{L}{2} = 0$

$\frac{L^3}{3} A_y - \frac{3L^3}{8} M_0 = 0$

$A_y = \frac{3}{L^3} \cdot \frac{3L^3}{8} M_0 = \frac{9M_0}{8L}$

$\therefore \vec{A} = \frac{9M_0}{8L} \uparrow$

We may assume that the shape of the deflected beam is sketched as shown. Under such an assumption, note that  $\vec{F}_{C/B} = \vec{C'C} \uparrow$ . The moment of  $\vec{F}_{C/B}$  with respect to the reference point B, where the tangent is drawn, is CLOCKWISE. Therefore, by the second moment-area theorem, we write

$t_{C/B} = +2 (M_C)_{C/B} = \text{moment, about point C, of the elastic weight between points C and B.}$

$t_{C/B} = -\frac{2}{3} \cdot \frac{L}{2} \cdot \frac{L}{4} \cdot \frac{L}{2EI} \cdot \frac{9M_0}{8L} - \frac{L}{4} \cdot \frac{L}{2} \cdot \frac{L}{2EI} \cdot \frac{9M_0}{8L} + \frac{L}{4} \cdot \frac{L}{2} \cdot \frac{M_0}{EI} = \frac{M_0 L^2}{128EI}$

$\vec{F}_{C/B} = \frac{M_0 L^2}{128EI} \uparrow$   $\vec{y}_c = \vec{C'C} = -t_{C/B} \therefore \vec{y}_c = \frac{M_0 L^2}{128EI} \downarrow$